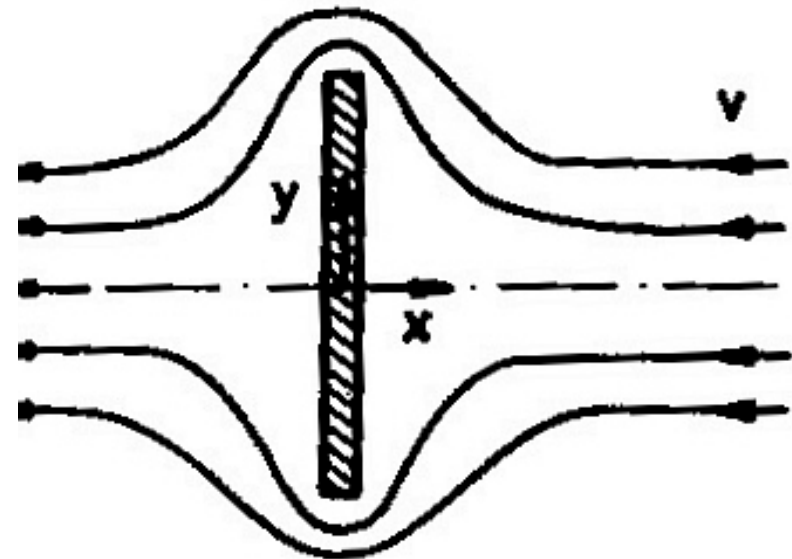
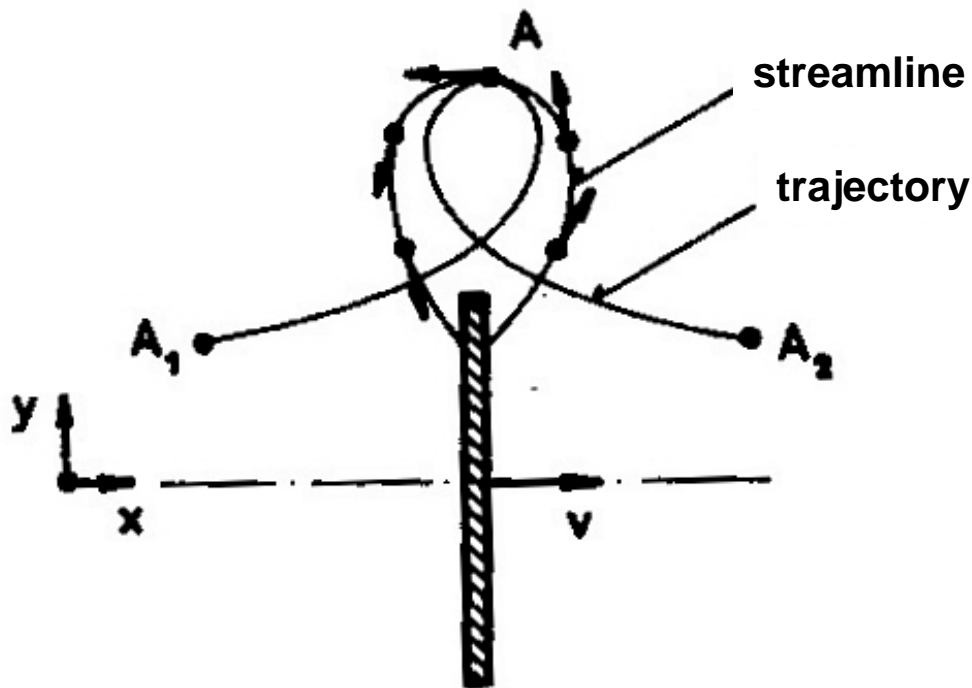


Fluid Mechanics



Fluid mechanics problems

fluids – liquids and gases
compressible, incompressible
flow – laminar
turbulent
ideal (perfect) fluid



flow velocity description – Lagrangian a Eulerian

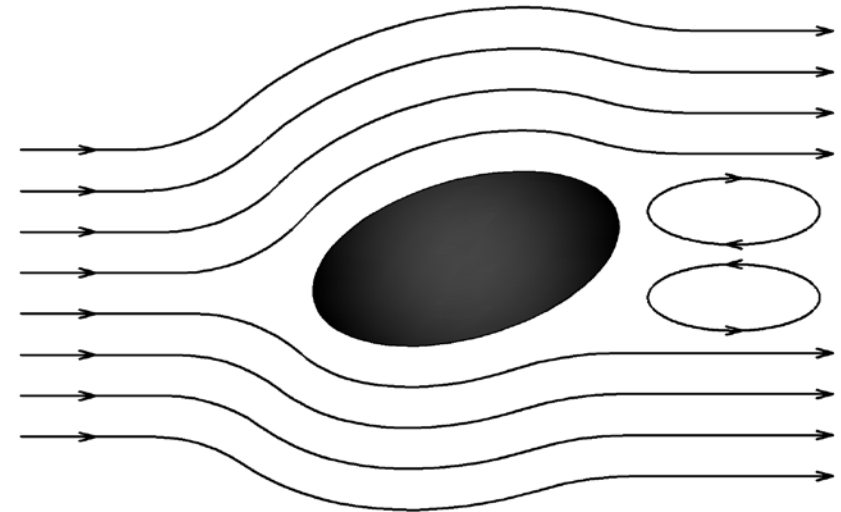
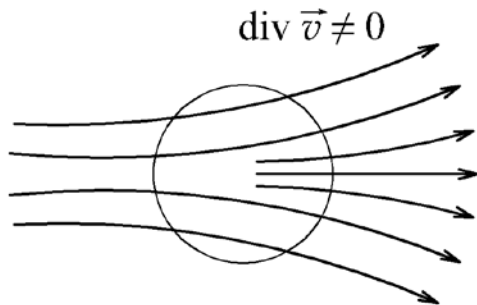
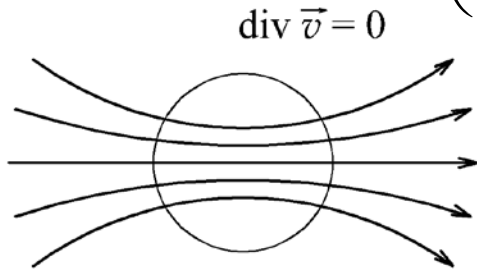
Streamlines

sources and sinks \rightarrow velocity divergence

$$\operatorname{div} \vec{v} = \vec{\nabla} \cdot \vec{v} \quad \vec{\nabla} = \left(\frac{\partial}{\partial x}, \frac{\partial}{\partial y}, \frac{\partial}{\partial z} \right)$$

$$\operatorname{div} \vec{v} = \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z}$$

$$\operatorname{rot} \vec{v} = \vec{\nabla} \times \vec{v}$$



$\operatorname{div} \vec{v} \neq 0$ **source / sink flow**

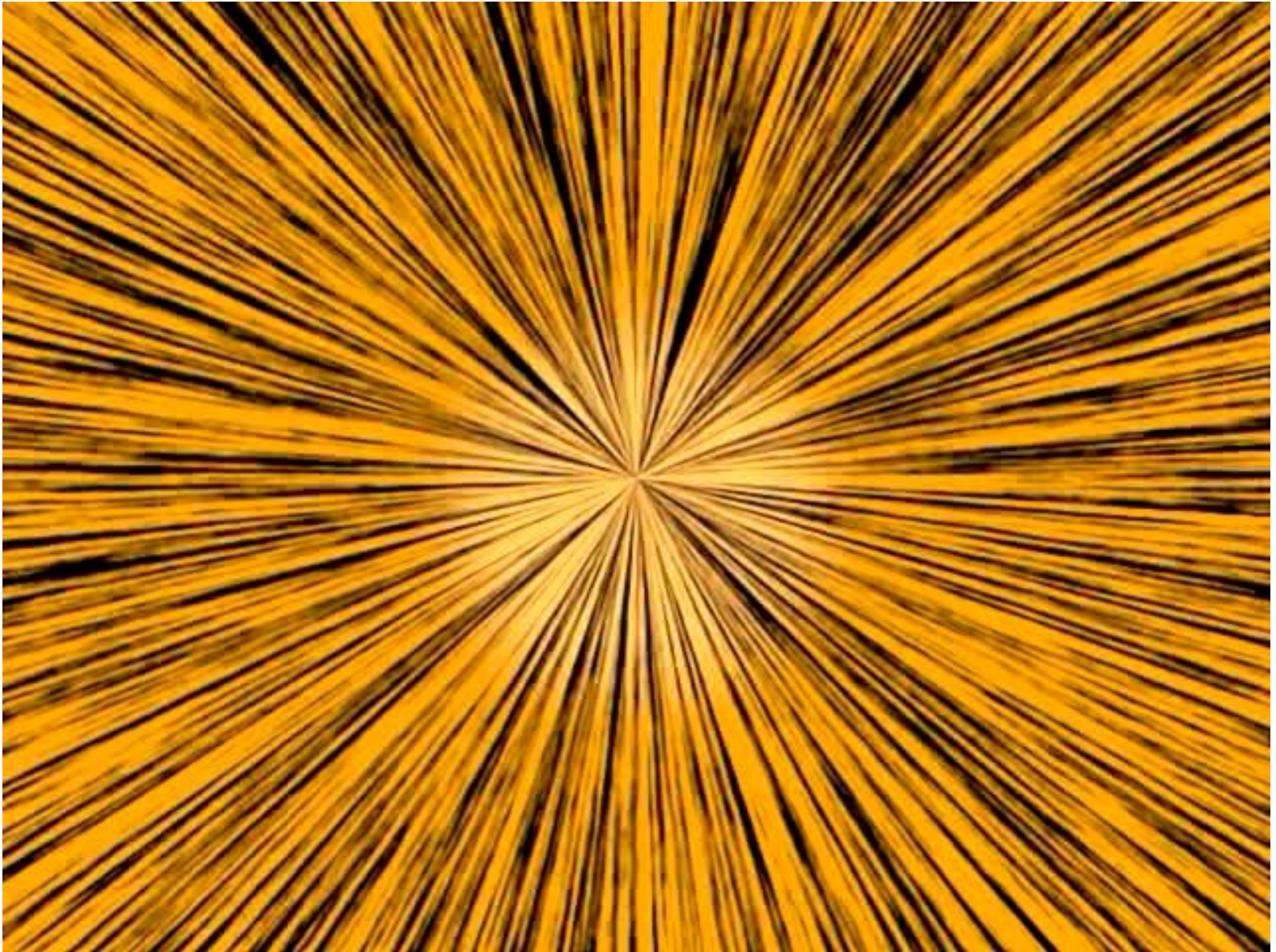
$\operatorname{rot} \vec{v} \neq 0$ **vorticity**

translation + rotation

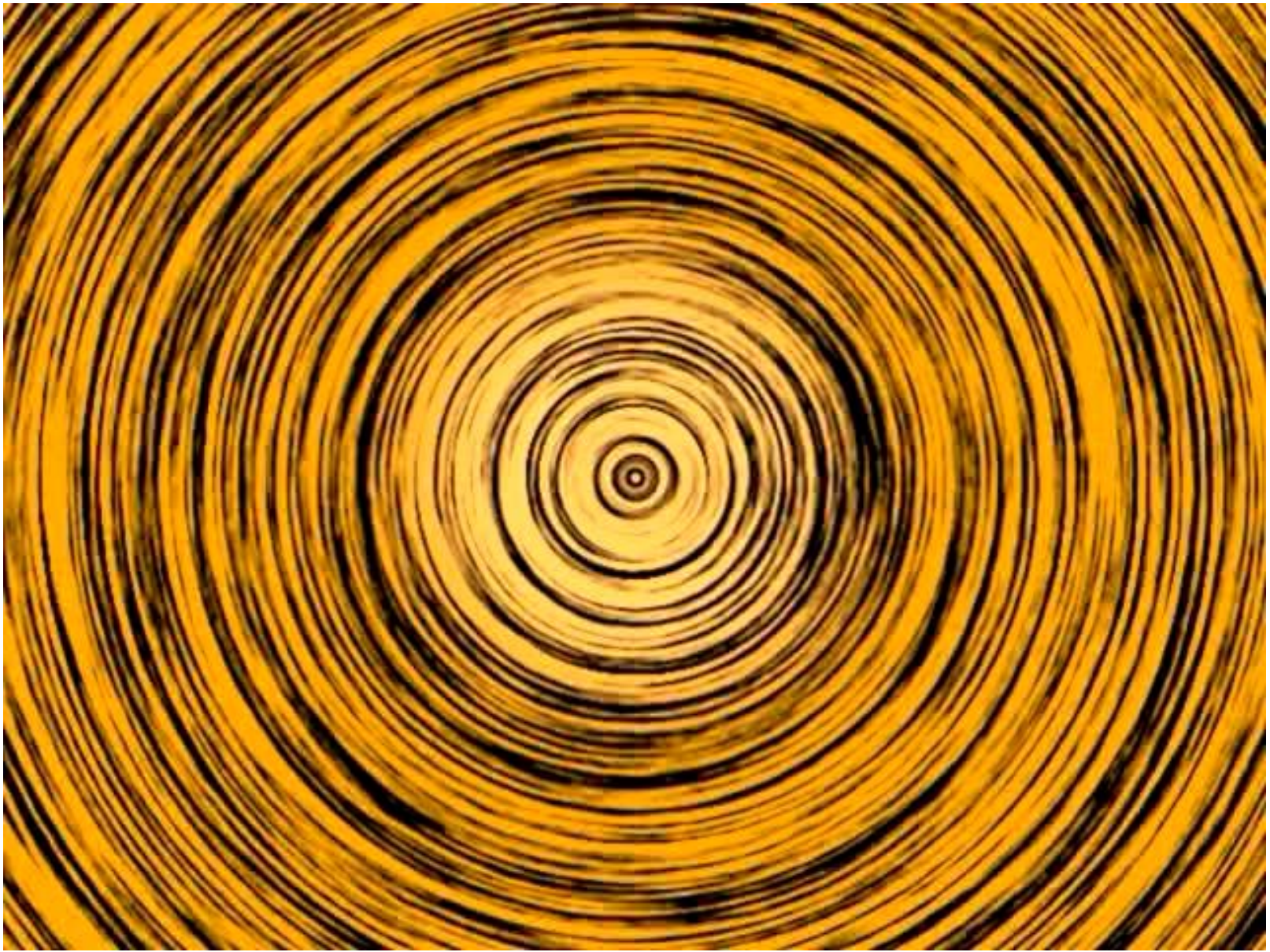
$$\vec{v} = \vec{v}_T + (\vec{\omega} \times \vec{r})$$

$$\operatorname{rot} \vec{v} = 2\vec{\omega}$$

$$\operatorname{rot} \vec{v} = \operatorname{rot} \vec{v}_T + \operatorname{rot} (\vec{\omega} \times \vec{r})$$



Fluid flow with a source



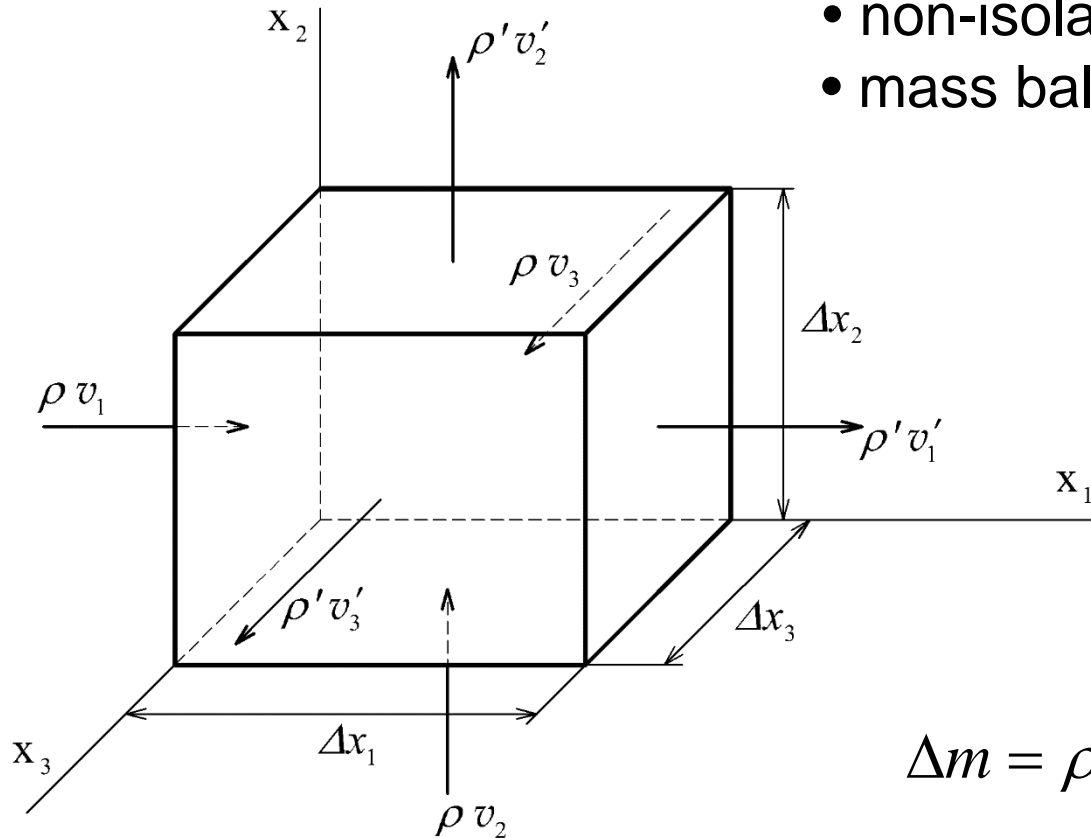
Fluid flow with a vortex



Fluid flow with source and vortex

Mass Balance Equation

- non-isolated system
- mass balance equation for ΔV



average mass flux

$$\vec{j}_m = \rho \vec{v}$$

$$\Delta m = \rho \Delta V$$

$$-\Delta m = [(\rho'v_1' - \rho v_1)\Delta S_1 + (\rho'v_2' - \rho v_2)\Delta S_2 + (\rho'v_3' - \rho v_3)\Delta S_3] \Delta t$$

conversion $\Delta m / (\Delta V \Delta t)$

$$-\frac{\Delta m}{\Delta V} \frac{1}{\Delta t} = \frac{\Delta(\rho v_1)}{\Delta x_1} + \dots$$

$$\operatorname{div} \rho \vec{v} = \sum_{k=1}^3 \frac{\partial (\rho v)_k}{\partial x_k} = \frac{\partial (\rho v_1)}{\partial x_1} + \frac{\partial (\rho v_2)}{\partial x_2} + \frac{\partial (\rho v_3)}{\partial x_3}$$

$$-\frac{\partial \rho}{\partial t} = \operatorname{div} \rho \vec{v}$$

Continuity Equation

$$\iiint_V \operatorname{div}(\rho \vec{v}) dV = - \iiint_V \frac{\partial \rho}{\partial t} dV$$

$$\oiint_S \rho \vec{v} \cdot d\vec{S} = \iiint_V \operatorname{div}(\rho \vec{v}) dV$$

$$\iiint_{\Delta V} \frac{\partial \rho}{\partial t} dV + \oiint_{\Delta S} \rho \vec{v} \cdot d\vec{S} = 0$$

An increase of mass dm within a volume dV equals to the difference of the inward mass flow and the outward mass flow.

Stationary (Steady) Flow

$$\frac{\partial \rho}{\partial t} = 0 \Rightarrow \operatorname{div} \rho \vec{v} = 0$$



$$\Delta m_1 = \rho_1 S_1 v_1 \Delta t$$

$$\rho_1 v_1 S_1 = \rho_2 v_2 S_2$$

$$\Delta m_2 = \rho_2 S_2 v_2 \Delta t$$

incompressible fluid

$$\frac{\partial \rho}{\partial t} = 0 \quad \operatorname{grad} \rho = 0$$

$$v_1 S_1 = v_2 S_2$$

$$\operatorname{div} \vec{v} = 0$$

$$\text{rot } \vec{v} = 2\vec{\omega} = \vec{\Omega}$$

vorticity

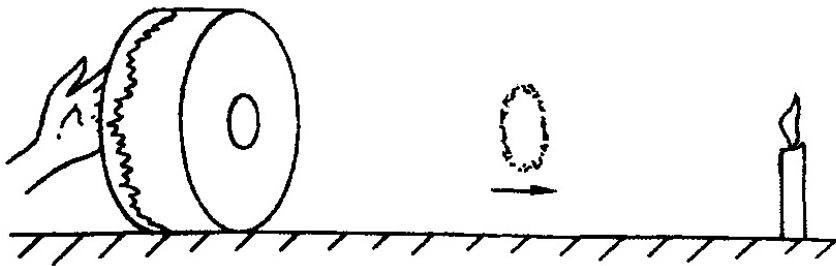
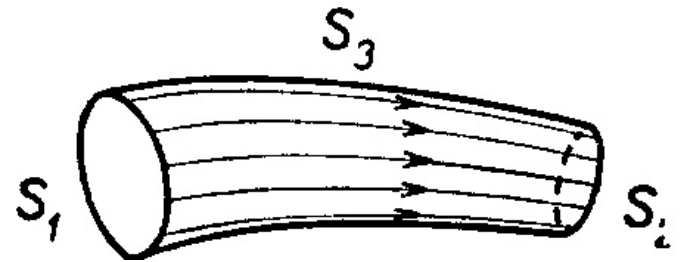
field of vortices

$$\text{div } \vec{\Omega} = 0 \quad \text{solenoidal field}$$

streamlines = vortex filament

stream tube = vortex tube

vortex intensity $\mu = \iint_S \vec{\Omega} \cdot d\vec{S}$



Balance Equations of Linear Momentum and Energy

$$\Delta m \vec{a} = \Delta \vec{F}_o + \Delta \vec{P}_t + \Delta \vec{P}_v$$

$$\lim_{\Delta V \rightarrow 0} \frac{\Delta m}{\Delta V} = \rho$$

$$\lim_{\Delta V \rightarrow 0} \frac{\Delta \vec{F}_o}{\Delta V} = \rho \vec{K}$$

$$\lim_{\Delta V \rightarrow 0} \frac{\Delta \vec{P}_t}{\Delta V} = -\text{grad } p$$

$$\lim_{\Delta V \rightarrow 0} \frac{\Delta \vec{P}_v}{\Delta V} = \vec{f}_v$$

$$\rho \vec{a} = \rho \vec{K} - \text{grad } p + \vec{f}_v$$

motion equation for a viscous fluid

$$\rho \vec{a} = \rho \vec{K} - \text{grad } p$$

Euler equation (non-viscous fluid)

$$\rho \vec{a} = -\rho \text{grad } U - \text{grad } p$$

$$\vec{a} \cdot d\vec{r} = -dU - \frac{1}{\rho} dp$$

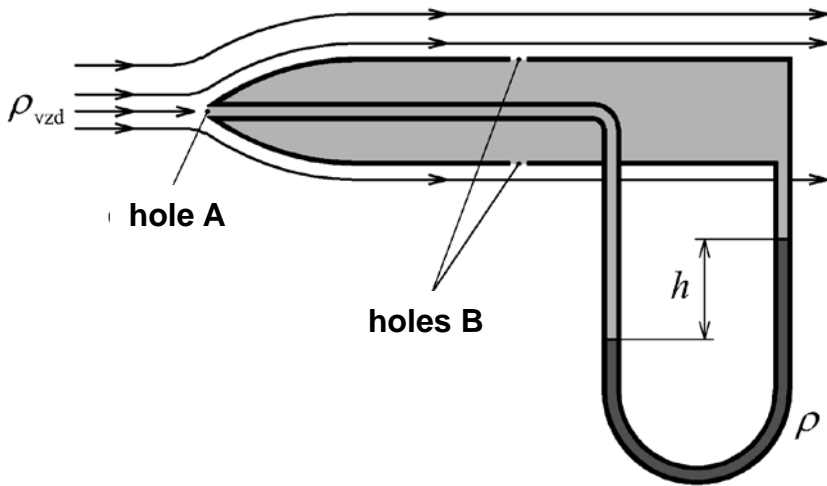
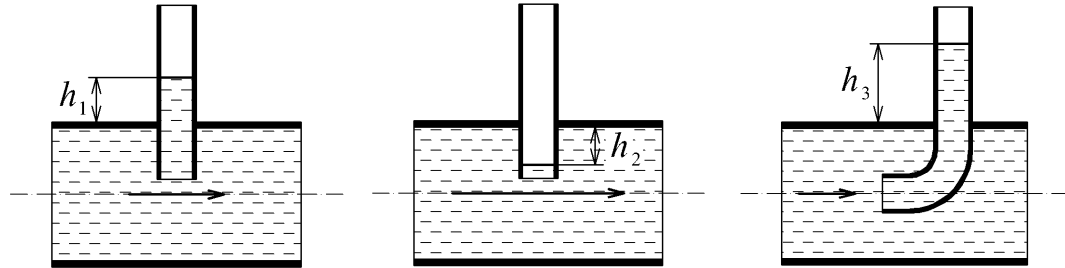
$$\vec{a} \cdot d\vec{r} = \sum_{i=1}^3 \frac{dv_i}{dt} dx_i = \sum_{i=1}^3 dv_i v_i = d\left(\frac{1}{2} v^2\right)$$

$$d\left(\frac{1}{2} v^2\right) + dU + \frac{1}{\rho} dp = 0$$

Bernoulli equation

$$\frac{1}{2} \rho v^2 + \rho U + p = \text{konst}$$

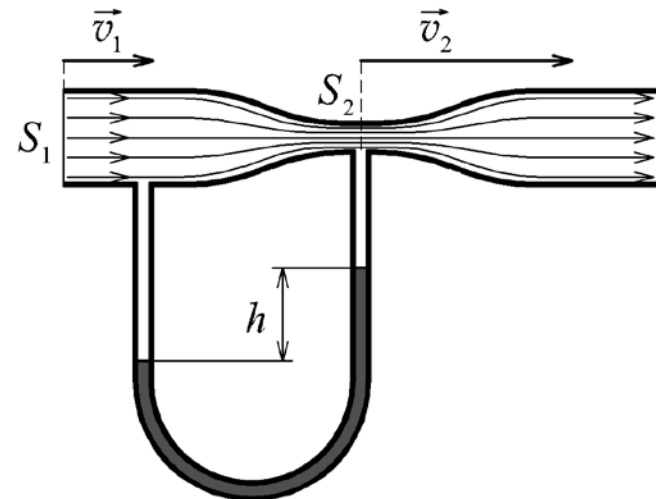
Pitot Tube



Pitot-static (Prandtl) tube

$$v = \sqrt{\frac{2\rho gh}{\rho_{vzd}}}$$

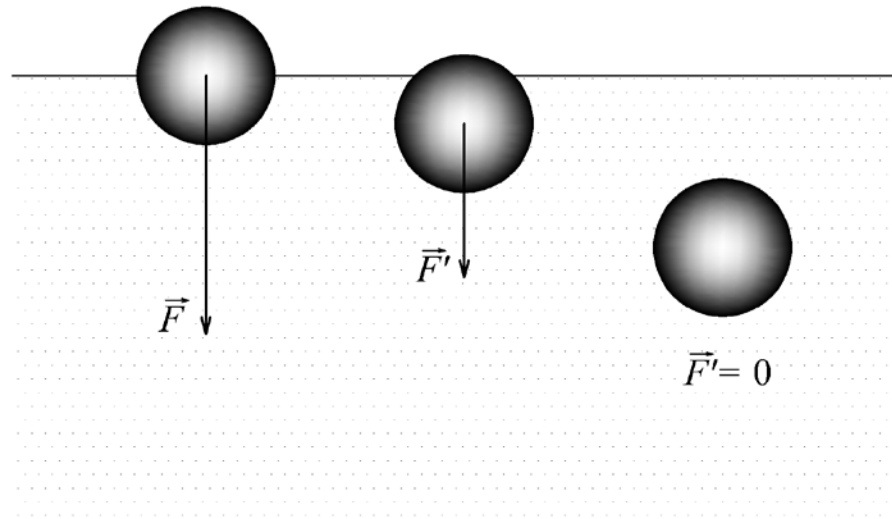
Venturi tube (flow meter)



$$v = \sqrt{\frac{2s^2 \Delta p}{\rho (S^2 - s^2)}}$$

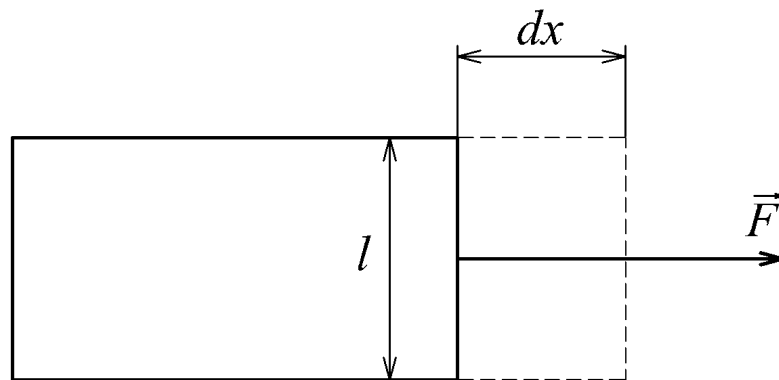
$$\Delta p = \rho_n gh$$

Surface Tension in Liquids



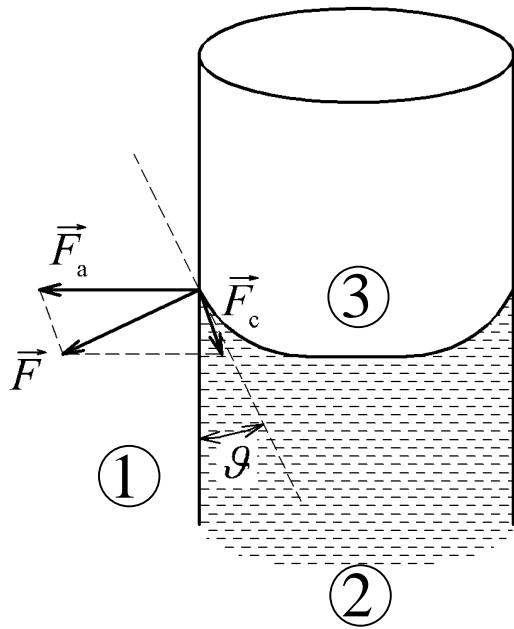
Surface tension equals to the work dA of the molecular interaction forces needed for a surface area dS generation.

$$\sigma = \frac{\partial A}{\partial S}$$



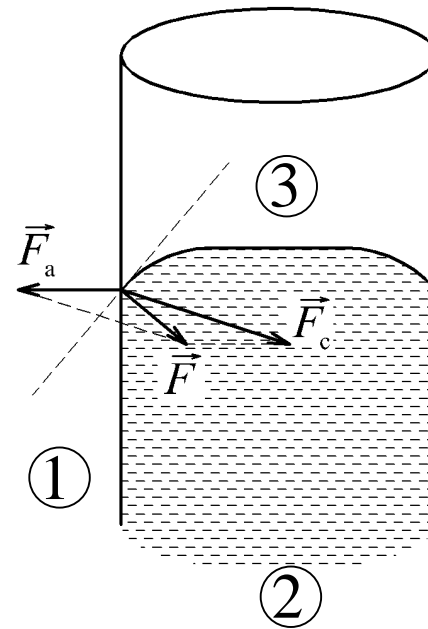
$$\sigma = \frac{F}{l}$$

$$ds = l dx$$



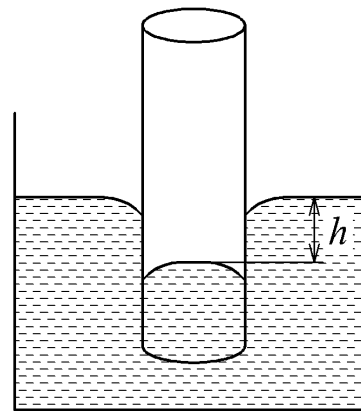
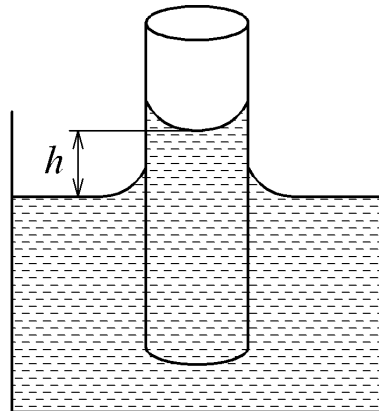
wetting

$$\sigma_{13} > \sigma_{12}$$

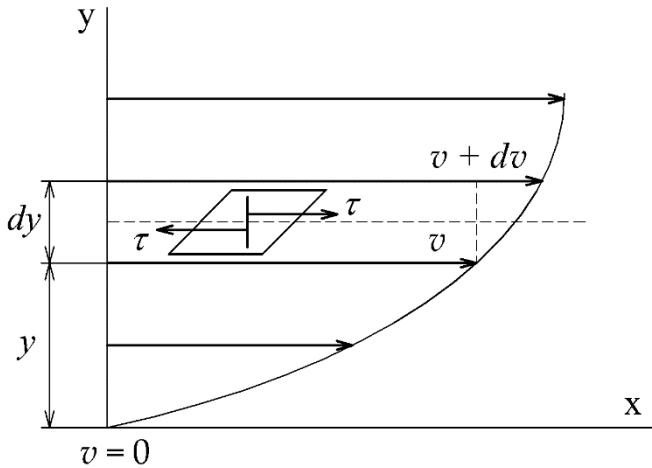


non-wetting

$$\sigma_{13} < \sigma_{12}$$

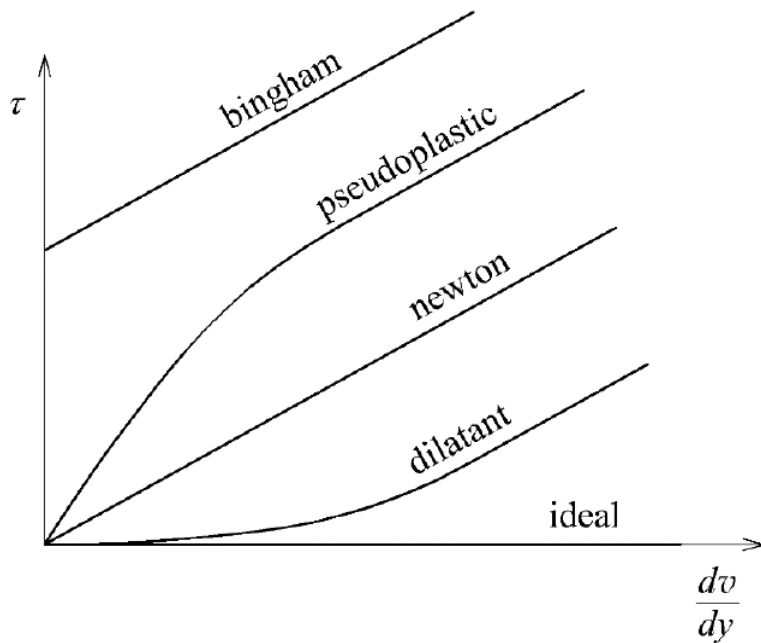


Viscosity



$$\tau = \eta \frac{dv}{dy} \quad \text{Newton's law of viscosity}$$

η dynamic viscosity coefficient



bingham – plaster, paste

pseudoplastic – wet sand

tixotropic – paints, yoghurt

$$\nu = \frac{\eta}{\rho} \quad \text{kinematic viscosity}$$